

Article

Assessment of Tribological Properties of Ti3C2 as a Water-Based Lubricant Additive

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Abstract: Water-based lubrication has attracted remarkable interest due to its environmental and economic advantages. However, practical applications of water-based lubrication are often limited, mainly because of low viscosity and corrosivity. The use of additives has been proposed to overcome these limitations. In this work, the tribological characteristics of titanium carbide ($Ti₃C₂$) MXenes, as additives for water-based lubrication, were systematically investigated for contact sliding between stainless steel under various normal forces and $Ti₃C₂$ concentrations. Both friction and wear were found to decrease with increasing $Ti₃C₂$ concentration up to 5 wt%, and then increased when the concentration was larger than 5 wt%. The results suggest that $Ti₃C₂$ flakes hindered direct contact, particularly at the edges of the contact interfaces. It was further shown that the agglomeration of Ti₃C₂ flakes may have reduced the hindering when an excessive amount of Ti₃C₂ (e.g., 7 wt%) was applied. The decreases in the friction coefficient and wear rate with 5 wt% of Ti₃C₂ concentration w approximately 20% and 48%, respectively. The outcomes of this work may be helpful in gaining a better understanding of the tribological properties of $Ti₃C₂$ as a feasible water-based lubrication additive.

Keywords: titanium carbide; friction; wear; water lubrication; lubricant additive

1. Introduction

Environmental degradation, such as resource depletion, climate change, and pollution, has become a growing concern in the last few decades. To overcome these problems, and to ensure a sustainable future, various green technologies have been under development [\[1–](#page-11-0)[3\]](#page-11-1). In particular, green tribology involves the minimization of friction and wear, environment-friendly or biodegradable lubrication, reduction or even elimination of lubrication, complete utilization of materials, biomimetic surface design, and tribology for renewable sources of energy, with the aim of saving energy and materials, and minimizing the harmful impacts on the environment and ecological balance [\[4](#page-11-2)[–6\]](#page-11-3). Among the green tribology solutions currently under development, water-based lubrication technology, which potentially can replace conventional oil-based lubricants, has attracted considerable interest due to its economic and environmental benefits, such as low cost, cleaning performance, natural resource conservation, and environmental sustainability.

Water-based lubrication is often limited in practical tribological applications, mainly due to its low viscosity and corrosivity. In order to overcome these limitations and improve the properties of water-based lubricants, extensive efforts have been made over the past decades. For example, fundamental studies to understand the performance of water-lubricated journal bearings have been performed [\[7,](#page-11-4)[8\]](#page-11-5). The tribological properties of various coatings with good lubricity in a water environment have also been investigated [\[9](#page-11-6)[,10\]](#page-11-7). In addition, to enhance the tribological properties of the coatings in water-based lubrication, surface texturing, such as nanostructure, micro-dimples, and grooves, has been proposed [\[11](#page-11-8)[–13\]](#page-11-9). Furthermore, investigations have been conducted on the use of additives to overcome the limitations of water-based lubrication [\[14\]](#page-11-10). Various nanoparticles,

including copper (Cu) [\[15\]](#page-11-11), diamond [\[16\]](#page-11-12), titanium dioxide (TiO₂) [\[17\]](#page-11-13), and silicon dioxide (SiO₂) [\[18\]](#page-11-14), have been proposed as potential candidates for lubricant additives. Carbon-based nanomaterials, such as fullerenes [\[19\]](#page-11-15) and carbon nanotubes [\[20\]](#page-12-0), have also attracted considerable interest as additives for water-based lubrication. Particularly, graphene and its derivatives have recently been demonstrated as promising lubricant additives, due to their low friction characteristics, associated with weak interatomic interactions between layers, and chemical inertness [\[21–](#page-12-1)[23\]](#page-12-2). For example, when graphene oxide was added to water, it was observed that the friction coefficient was maintained at approximately 0.05 for up to 60,000 cycles, without any significant wear [\[22\]](#page-12-3). It was demonstrated that the decrease in friction and wear may have been associated with the formation of a tribo-film that can act as a protective coating [\[22,](#page-12-3)[23\]](#page-12-2). Furthermore, considering that other two-dimensional (2D) layered materials, such as molybdenum disulfide $(MoS₂)$ and hexagonal-boron nitride (h-BN), can provide low-frictional properties [\[24](#page-12-4)[–26\]](#page-12-5), their tribological characteristics as additives for water-based lubrication have been explored [\[27,](#page-12-6)[28\]](#page-12-7).

Recently, 2D layered transition metal carbides and nitrides, MXenes, have attracted attention as candidates for solid lubricants [\[29](#page-12-8)[,30\]](#page-12-9). The general structure of MXenes is $M_{n+1}X_nT_x$, where M is an early transition metal, X is C and/or N, and T represents a functional group. Titanium carbide (Ti_3C_2) , often denoted as $Ti_3C_2T_x$, where T is F, OH, and O, is one of the most extensively studied MXenes due to its potential for various applications [\[29\]](#page-12-8). It was shown that Ti_3C_2 may provide a significant reduction of friction and wear, attributed to the prevention of direct contact and the formation of a carbon-rich tribo-film at the contact interface [\[31\]](#page-12-10). The tribological properties of $Ti₃C₂$ were further correlated with surface terminations and intercalated water [\[32\]](#page-12-11). In particular, a practical approach for the use of $Ti₃C₂$ as a solid lubricant for thrust ball bearings was conducted [\[33\]](#page-12-12). Furthermore, it was demonstrated that $Ti₃C₂$ can be exploited as a lubricant additive to oil- and water-based lubricants [\[34](#page-12-13)[–38\]](#page-12-14). However, more data should be accumulated on utilizing Ti_3C_2 as a lubricant additive.

In this work, the tribological properties of $Ti₃C₂$ as an additive in water-based lubrication were experimentally investigated using a ball-on-disk tribotester. Experiments were performed using a stainless steel (SS) ball and disk at boundary lubrication, under various normal forces and Ti_3C_2 concentrations in water. The variation in friction was monitored during the tests and the wear rates of the specimens were quantitatively assessed using a laser scanning confocal microscope (LSCM) after the experiments. A better understanding of the wear behavior with $Ti₃C₂$ additives was additionally pursued using scanning electron microscope (SEM) observations. The outcomes may provide useful information to understand the effect of $Ti₃C₂$ on the tribological characteristic of SS under water-based lubrication.

2. Materials and Methods

2.1. Materials

Commercialized Ti₃C₂ flakes with 99.9% purity (Wuhan Golden Wing Industry & Trade, Wuhan, China) were used in this work. The nominal size of the flakes was $0.2-3 \mu m$. Figure [1a](#page-2-0) shows SEM images of Ti₃C₂ flakes. It can be seen from the figure that the lateral size of Ti₃C₂ flakes varied from less than 1 μm to several μm. The high-magnification SEM image inset in Figure [1a](#page-2-0) shows clearly the layered structure of Ti_3C_2 . In addition, the thickness of the layer was found to be approximately 20 nm, which was consistent with the thickness reported in the literature [\[34](#page-12-13)[,37\]](#page-12-15). The X-ray diffraction (XRD) pattern of the flakes is also provided in Figure [1a](#page-2-0), indicating the formation of Ti_3C_2 [\[39\]](#page-12-16). Five solutions with different concentrations of Ti_3C_2 in pure deionized (DI) water were prepared, with the aim of investigating the effect of concentration on friction and wear characteristics. To prepare the solutions, 0.01 g, 0.02 g, 0.03 g, 0.05 g, and 0.07 g of Ti_3C_2 were added to 1 mL of DI water for the solutions with 1 wt%, 2 wt%, 3 wt%, 5 wt%, and 7 wt% $Ti₃C₂$ concentrations, respectively. Then, each solution was stirred in a magnetic stirrer for 1 h at room temperature to ensure that the $Ti₃C₂$ was uniformly dispersed. Figure [1b](#page-2-0) shows photographs of the DI water and the DI water solutions with five different α concentrations of Ti₃C₂ additives.

Figure 1. (a) SEM images and XRD pattern of Ti_3C_2 flakes, (b) photographs of water-based lubricants with 0 wt%, 1 wt%, 2 wt%, 3 wt%, 5 wt%, and 7 wt% Ti_3C_2 concentrations, and photographs and 3D laser scanning confocal microscope (LSCM) images of the (c) ball and (d) disk. Inset in (a) is a high-magnification of the SEM image.

Martensitic SS (AISI 440C) was selected as the ball material due to its high hardness and Martensitic SS (AISI 440C) was selected as the ball material due to its high hardness and corrosion resistance in water. The nominal radius of the balls was 3 mm. All balls used in this work were carefully examined under a LSCM (VK-X200, Keyence, Osaka, Japan) to determine the radii, and to assess their surfaces before experiment. [Fi](#page-2-0)gure 1c shows a photograph of the ball glued on the holder for the experiment, along with an example of a three-dimensional (3D) LSCM image before experiment. The average radius of the balls was calculated to be 3.02 ± 0.01 mm (mean \pm one standard deviation). It can be seen [fro](#page-2-0)m Figure 1c that the surface of the ball before experiment was quite clean. The average surface roughness (R_a) of the balls was also determined from the LSCM data with a scan size of 1270 μ m \times 997 µm after the flattening process. The R_a value of the balls was calculated to be 0.29 \pm 0.01 µm. The hardness of the balls was further determined using a micro-Vickers hardness tester (MMT-7, Matsuzawa, Japan). To that end, a ball was ground to form a flat surface. Hardness measurement was performed with an indentation force of 0.98 N and dwell time of 20 s at five different randomly selected locations on three balls. The hardness values of the three balls were found to be similar to one another (765 \pm 28 HV, 785 \pm 34 HV, and 730 \pm 10 HV). The average hardness value was determined to be 760 \pm 23 HV. The hardness values of the balls determined in this work were in good agreement with those of AISI 440C SS.

Austenitic SS (AISI 304) was used as the disk material. SS disks with a radius of 40 mm and a Austenitic SS (AISI 304) was used as the disk material. SS disks with a radius of 40 mm and a thickness of 10 mm were prepared for the experiment. Figure [1d](#page-2-0) shows the photograph and 3D LSCM thickness of 10 mm were prepared for the experiment. Figure 1d shows the photograph and 3D LSCM image of a disk before experiment. From Figure [1d](#page-2-0), the patterns, which likely formed during the manufacturing process, can be observed. The R_a value of the disks was determined to be 1.67 ± 0.05 μ m from the LSCM data, with a scan size of 1408 μ m \times 1056 μ m obtained at five different locations on five different disks. The hardness of the disks was measured from five different locations of five disks using the micro-Vickers hardness tester (MMT-7, Matsuzawa, Japan), with an indentation force of 0.98 N and dwell time of 20 s. The average hardness value of the disks was 220 ± 10 HV, which was within the typical range of hardness of AISI 304 SS.

2.2. Methods

The tribological properties of Ti_3C_2 as a water-based lubricant additive for contact sliding between a SS ball and a disk were investigated using a ball-on-disk tribotester under boundary lubrication. A photograph of the tester used in this work is shown in Figure [2.](#page-3-0) A normal force was applied by a dead weight, and the friction force was monitored using a load cell equipped with the tester. The experiments were conducted for a specimen radius of 10 mm, therefore the sliding distance of 1 cycle was 0.063 m. The rotating speed was set to 120 rpm, which corresponds to a linear sliding speed of 0.126 m/s. The normal force varied from 3 N to 10 N, which corresponded to a contact pressure from 0.91 to 1.36 GPa, as calculated using Hertzian contact model [\[40\]](#page-12-17). The experiment with a given normal force was conducted for up to 7000 cycles. A 0.2 mL quantity of lubricant was applied to the contact area using a syringe immediately after preparation, to minimize the effect of dispersibility of additives on tribological performance. The experiments were repeated at least three times for each experimental condition. Prior to the experiments, the balls and disks were cleaned with isopropyl alcohol (IPA) by ultrasonication for 1 h. All tests were performed in ambient conditions with the temperature 24 °C–26 °C and relative humidity 37–46%. with the temperature 24 °C–26 °C and relative humidity 37–46%.

Figure 2. Photograph of the ball-on-disk tribotester used in this work. **Figure 2.** Photograph of the ball-on-disk tribotester used in this work.

The surface of the balls and the wear track formed on the disks were carefully examined after experiments using the LSCM to understand the wear characteristics of water-based lubricants with experiments using the LSCM to understand the wear characteristics of water-based lubricants with $Ti₃C₂$ additives. In particular, the wear rates of the ball and disk were carefully determined using $I₃C₂C₁$ LSCM data. To that end, additional LSCM images of the balls and disks were acquired after ultrasonic cleaning, since a significant amount of wear debris and additives may have been readily deposited cleaning, since a significant amount of wear debris and additives may have been readily deposited on the surfaces during the experiment. The wear volume of the balls was determined as the volume of the spherical segment calculated using the cross-sectional height profiles at the center of the ball $\frac{1}{2}$ before and after experiment, considering that the ball was flattened due to wear. As for the disks, before and after experiment, considering that the ball was flattened due to wear. As for the disks, LSCM images were obtained at five different locations of the wear track, and averaged cross-sectional LSCM images were obtained at five different locations of the wear track, and averaged cross-sectional height profiles were taken from each image. Then, the wear volume of the disk was calculated by multiplying the average cross-sectional area of the wear track by the circumferential contact multiplying the average cross-sectional area of the wear track by the circumferential contact length. Furthermore, SEM observations and energy-dispersive X-ray spectroscopy (EDS) analyses were length. Furthermore, SEM observations and energy-dispersive X-ray spectroscopy (EDS) analyses were performed, particularly of the Ti_3C_2 flakes after the experiment, to gain a better understanding of tribological behavior of water-based lubricants with Ti_3C_2 additives. The surface of the balls and the wear track formed on the disks were carefully examined after LSCM data. To that end, additional LSCM images of the balls and disks were acquired after ultrasonic on the surfaces during the experiment. The wear volume of the balls was determined as the volume height profiles were taken from each image. Then, the wear volume of the disk was calculated

3. Results and Discussion $\overline{3000}$ coefficient became relatively stable, and ranged from 0.24, as the $\overline{340}$ $T₁$ concentration varied from α with α with α is planet varies va

3.1. Friction Characteristics i in the variation coefficient during the run-in period could be observed, which was likely which was likely

Figure [3a](#page-4-0) shows the variation of friction coefficient with the number of cycles under a 3 N normal force, obtained using the water lubricants with 0 wt%, 1 wt%, 2 wt%, 3 wt%, 5 wt%, and 7 wt% Ti_3C_2 concentrations. The initial friction coefficient was found to range from 0.17 to 0.31. Additionally, it was shown that the friction coefficient fluctuated during the initial stage of contact sliding for up to 3000 cycles. Then, the friction coefficient became relatively stable, and ranged from 0.24 to 0.34, as the Ti₃C₂ concentration varied from 0 wt% to 7 wt%. It is plausible that the friction coefficient varies during the run-in process, and becomes stable with increasing the number of cycles [\[41\]](#page-12-18). A difference in the variation of friction coefficient during the run-in period could be observed, which was likely associated with the initial states of ball and disk surfaces, and Ti $_3C_2$ flakes at the contact interface. The difference in these initial states might have further led to the difference in duration of the run-in period. The data presented in Figure [3a](#page-4-0) show that the friction coefficients for the water lubricants with Ti_3C_2 additives were consistently lower than that without Ti_3C_2 additives. It can also be seen from the figure that the decrease in friction coefficient was the largest when the Ti_3C_2 concentration was 5 wt%. $f(x) = \frac{1}{\sqrt{2}} \int_0^x \frac{dx}{\sqrt{2}} dx$ is decreased in friction coefficient was the largest when the $\frac{1}{\sqrt{2}} \int_0^x \frac{1}{2\sqrt{2}} dx$ α approach to be approximately α and the that for α of α α β concentration.

Figure 3. (a) Variation of friction coefficient with the number of cycles under 3 N normal force, (**b**) variation of friction coefficient at steady state with T_3C_2 concentration under 3 N–10 N normal force, and (c) variation of average friction coefficient with $Ti₃C₂$ concentration. Inset in (a) is the plot of friction coefficient for the number of cycles below 200. friction coefficient for the number of cycles below 200.

The friction coefficients were averaged after they became stable (i.e., from 3000 to 7000 cycles) for a clear comparison, and the variation of average friction coefficient was plotted against the Ti_3C_2 concentration under various normal forces, as shown in Figure [3b](#page-4-0). It can be observed from the figure $\frac{1}{2}$ which is a weakly the frictions, respectively, concentration coefficient for a 0.5×10^{10} that the friction coefficient generally decreased as the Ti_3C_2 concentration increased up to 5 wt%, but it increased when the Ti $_3$ C₂ concentration was 7 wt%, as anticipated from the data in Figure [3a](#page-4-0). However, the effect of normal force on friction coefficient was not clearly observed. The friction coefficients for 0 wt% and 5 wt% Ti $_3$ C₂ concentrations were found to range from 0.29 to 0.33, and from 0.20 to 0.26, respectively. The decrease in the friction coefficient for a 5 wt% Ti $_3\rm{C}_2$ concentration was calculated to be approximately 13–31%, compared to that for a 0 wt% Ti $_3\mathrm{C}_2$ concentration.

The friction coefficients obtained for normal forces ranging from 3 N to 10 N were further averaged to investigate more carefully the effect of $Ti₃C₂$ concentration, as shown in Figure [3c](#page-4-0). It was found that the friction coefficients decreased by 7%, 7%, 17%, 20%, and 10%, for 1 wt%, 2 wt%, 3 wt%, 5 wt%, $\frac{2}{3}$, $\frac{20}{\pi}$; $\frac{22}{\pi}$, the decrease in the friction provided by Ti3C2 was comparable with $\frac{1}{2}$ wi and 7 wt% Ti₃C₂ concentrations, respectively, compared to the friction coefficient for a 0 wt% Ti₃C₂ concentration. These observations clearly suggest that Ti_3C_2 can reduce friction between a SS ball and a SS disk as an additive for water-based lubrication. The decrease in friction may be associated with behavior of the Ti_3C_2 (e.g., adherence to surface and trapping in grooves) that enters into the contacting interface, as proposed in previous studies [\[34–](#page-12-13)[37\]](#page-12-15). Furthermore, it can be concluded that the optimal concentration of Ti_3C_2 for friction reduction was 5 wt%, for the considered material pair and experimental conditions adopted for this work. The friction reduction effect may likely decrease when an excess Ti_3C_2 is applied to the contact interface (e.g., Ti_3C_2 concentrations larger than 5 wt%), as consistently observed in previous investigations [\[34](#page-12-13)[–37\]](#page-12-15). The degree of friction reduction by Ti_3C_2 for water-based lubrication was relatively smaller than that offered by graphene-based materials [\[22](#page-12-3)[,23\]](#page-12-2). However, the decrease in the friction provided by Ti_3C_2 was comparable with those offered by other 2D materials, such as h-BN $[27]$ and MoS₂ $[28]$, in water-based lubrication. Interestingly, the friction reduction obtained with Ti_3C_2 for oil-based lubrication [\[34–](#page-12-13)[37\]](#page-12-15) is generally smaller than when using graphene-based materials $[42,43]$ $[42,43]$, but comparable to those offered by h-BN $[44]$ and MoS₂ [\[45\]](#page-13-2). It should further be noted that the degree of friction reduction by additives may significantly vary depending on experimental conditions. Nevertheless, the overall results in Figure [3](#page-4-0) clearly suggest the feasibility of using Ti $_3$ C₂ as an additive for water-based lubrication.

3.2. Wear Characteristics \overline{a} clearly support the feasibility of using Ti3C2 as an additive for water-based lubrication.

The wear characteristics of the balls and disks were assessed to elucidate further the feasibility of *3.2. Wear Characteristics* Ti $_3\rm{C}_2$ as a water-based lubrication additive. Figure 4 a shows 3D LSCM images of the balls cleaned using IPA after the experiments under a 3 N normal force. As examples, images of the balls tested using the water lubricants with 0 wt%, 1 wt%, and 5 wt% $Ti₃C₂$ concentrations are shown in Figure [4a](#page-5-0). In Figure [4a](#page-5-0), the cross-sectional height profiles before and after the experiment are provided for clear \overrightarrow{a} observation of wear. It can be seen from the figure that the balls were generally flattened due to observation of wear. It can be seen from the figure that the balls were generally flattened due to
the wear. In addition, it was observed that slight scratches appeared on the ball surface along the sliding direction. sliding direction. the wear. In addition, it was observed that slight scratches appeared on the ball surface along

Figure 4. (a) 3D LSCM images and cross-sectional height profiles of balls after experiments with 0 wt%, 1 wt%, and 5 wt% Ti_3C_2 concentrations under 3 N normal force, (**b**) variation of ball wear volume and (c) wear rate under 3 N–10 N normal force with Ti3C2 concentration, and (d) variation of average wear rate with Ti₃C₂ concentration. In (**a**), cross-sectional profiles before experiments are included for comparison.

The ball wear volumes and wear rates for various normal forces are plotted as functions of Ti_3C_2 concentration in Fig[ure](#page-5-0) 4b,c, respectively. The ball wear volume ranged from (6.4 ± 1.6) × 10⁻⁴ mm³ $(7 + 1.6) \times 10^{-3}$ mm³ under 3 N–10 N normal forces with 0 wt%–7 wt%. Tisca concentrations. to $(3.7 \pm 1.6) \times 10^{-3}$ mm³ under 3 N–10 N normal forces with 0 wt%–7 wt% Ti₃C₂ concentrations.

No clear effect of the normal force on the ball wear volume was observed in Figure [4b](#page-5-0). Additionally, it was shown that the effect of Ti_3C_2 concentration on the wear volume was not significant under a given normal force. The ball wear rate was calculated to be from $(1.0 \pm 0.3) \times 10^{-8}$ mm³/(N·cycle) to $(13 \pm 3.7) \times 10^{-8}$ mm³/(N·cycle) under 3 N–10 N normal forces with 0 wt%–7 wt% Ti₃C₂ concentrations, as shown in Figure [4c](#page-5-0). As predicted from the data shown in Figure [4b](#page-5-0), no significant dependence of the wear rate on the normal force and $Ti₃C₂$ concentration was further observed. This result suggests that the ball wear did not progress enough for the effects of normal force and $Ti₃C₂$ concentration to be observed clearly. For clarity, the variation of average ball wear rate with $Ti₃C₂$ concentration is plotted in Figure [4d](#page-5-0). The ball wear rates with 1 wt%–7 wt% $Ti₃C₂$ concentrations were slightly higher than with 0 wt% Ti₃C₂ concentration, within the experimental uncertainties. As expected, the effect of Ti₃C₂ concentration on the ball wear rate was not significant. This result might be feasibly associated with the higher hardness of the ball than that of the disk, which resulted in the suppression of ball wear.

A long-term test would be preferable for clear observation of the effects of normal force and Ti_3C_2

concentration on the ball wear. Figure [5a](#page-7-0) shows the LSCM images of the wear tracks formed on the disks after the experiments under a 3 N normal force for varying $Ti₃C₂$ concentrations. The averaged cross-sectional height profiles of the wear tracks are also presented in Figure [5a](#page-7-0). The data in Figure [5a](#page-7-0) were obtained after the IPA cleaning. The images in Figure [5a](#page-7-0) demonstrate clearly that a significant amount of scratches were formed along the sliding direction on all disks. It was observed from the data in Figure [5a](#page-7-0) that in general, the effect of Ti_3C_2 concentration on the depth of the wear track was not consistent (depths of 1.64 μ m, 0.6 μ m, 1.49 μ m, 0.52 μ m, 0.50 μ m, and 0.68 μ m, for 0 wt%, 1 wt%, 2 wt%, 3 wt%, 5 wt%, and 7 wt% $Ti₃C₂$ concentrations, respectively). However, it can clearly be seen that the width of the wear track significantly decreased as the $Ti₃C₂$ concentration increased up to 5 wt% (widths of 452.0 µm, 371.2 µm, 340.2 µm, 275.0 µm, and 236.6 µm, for 0 wt%, 1 wt%, 2 wt%, 3 wt%, and 5 wt% $Ti₃C₂$ concentrations, respectively). Subsequently, the width of the wear track was found to increase (341.3 μ m) when the Ti₃C₂ concentration increased to 7 wt%. The width of the wear track was expected to increase as the flattened area of the ball increased. However, no significant relationship between the flattened width of the ball and the width of the wear track on disk was observed from Figures [4a](#page-5-0) and [5a](#page-7-0). For example, although the flattened width of the ball after the experiment with a 7 wt% Ti_3C_2 concentration was larger than that after the experiment with 0 wt% Ti₃C₂ (Figure [4a](#page-5-0)), the wear track was wider for 0 wt% Ti_3C_2 concentration than for 7 wt% Ti_3C_2 concentration (Figure [5a](#page-7-0)). These results indicate that the direct contact between the ball and disk may have been hindered by the Ti_3C_2 flakes, particularly at the edges of the contact interface, and that such hindering by $Ti₃C₂$ flakes was more substantial as Ti₃C₂ concentration increased up to 5 wt%. However, it was postulated that this behavior of Ti₃C₂ flakes was reduced when an excessive amount of Ti₃C₂ flakes (e.g., 7 wt%) was applied to the contact interface.

Figure 5. (a) LSCM images and cross-sectional height profiles of disks after experiments with 0 wt%, 1 w_t 2 wt = wt = wt = wt = wt = concentrations under 3 N normal force, (**b**) variations under 3 N normal force, (**b**) variation of α 1 wt\% , 2 wt\% , 3 wt\% , 5 wt\% , and 7 wt\% Ti_3C_2 concentrations under 3 N normal force, (**b**) variation of disk wear volume and (**c**) wear rate under 3 N–10 N normal force with Ti₃C₂ concentration, and (d) variation of average wear rate with $Ti₃C₂$ concentration.

The variation of disk wear volume with $Ti₃C₂$ concentration is presented in Figure [5b](#page-7-0). The wear the dislections of from $(1.0 + 0.0) \times 10^{-2}$ with $(5.1 + 0.2) \times 10^{-2}$ with ϵ in ϵ volume of the disks ranged from $(1.9 \pm 0.6) \times 10^{-2}$ mm³ to $(5.1 \pm 0.3) \times 10^{-2}$ mm³ as the normal force varied from 3 N to 10 N and the Ti_3C_2 concentration from 0 wt% to 7 wt%. Similarly to the wear of the balls, the effect of normal force on the disk wear volume was not clearly observed. However, it can be seen that the disk wear volume generally decreased as the $Ti₃C₂$ concentration increased to 5 wt%, and then increased when $Ti₃C₂$ concentration was 7 wt%, as expected from the data presented in Figure [5a](#page-7-0). The decrease in the wear volume for 5 wt% Ti₃C₂ concentration (1.9 × 10⁻² mm³) was as high as 55% under 3 N of normal force, compared to the wear volume for 0 wt% Ti_3C_2 concentration $(4.3 \times 10^{-2} \text{ mm}^3)$.

Figure [5c](#page-7-0) shows the variation of disk wear rate for a given normal force with $Ti₃C₂$ concentration. It can be seen in the figure that the wear rate decreased with increasing normal force. For example, as the normal force increased from 3 N to 10 N, the wear rates for 0 wt% and 5 wt% Ti₃C₂ concentrations decreased from $(20 \pm 3.5) \times 10^{-7}$ mm³/(N·cycle) to $(6.7 \pm 0.9) \times 10^{-7}$ mm³/(N·cycle), and from (9.2 ± 2.9) \times 10⁻⁷ mm³/(N·cycle) to (3.4 ± 0.7) \times 10⁻⁷ mm³/(N·cycle), respectively. The Ti₃C₂ flakes may have been more damaged due to interactions with the ball, disk, and other $Ti₃C₂$ flakes under higher normal force. However, this may contribute to enhanced formation of a tribo-film that can suppress wear progression [\[34](#page-12-13)[–37\]](#page-12-15), which could be one of the reasons for the low wear rate under high normal force. In general, the wear rate was found to decrease with increasing $Ti₃C₂$ concentration from 0 wt% to 5 wt%, and then increase when Ti_3C_2 concentration was higher than 5 wt%. To clearly examine the

effect of Ti_3C_2 concentration on the disk wear rate, the average wear rates for a given concentration are shown in Figure [5d](#page-7-0). The wear rates were calculated to be (12 ± 1.7) \times 10^{-7} mm³/(N·cycle), (9.6 ± 0.8) \times 10^{-7} mm³/(N·cycle), $(7.9 \pm 1.9) \times 10^{-7}$ mm³/(N·cycle), $(7.8 \pm 2.0) \times 10^{-7}$ mm³/(N·cycle), $(6.2 \pm 1.5) \times$ 10^{-7} mm³/(N·cycle), and $(9.2 \pm 2.4) \times 10^{-7}$ mm³/(N·cycle) for Ti₃C₂ concentrations of 0 wt%, 1 wt%, 2 wt%, 3 wt%, 5 wt%, and 7 wt%, respectively. The decrease in wear rates for 1 wt%, 2 wt%, 3 wt%, 5 wt%, and 7 wt% Ti3C² concentrations was calculated to be 20%, 34%, 35%, 48%, and 23%, respectively, compared to the wear rate for 0 wt% $Ti₃C₂$ concentration. The decrease in the disk wear rate for 5 wt% $Ti₃C₂$ concentration was relatively large, compared to that provided by h-BN [\[27\]](#page-12-6), while it was relatively small, compared to that offered by graphene-based materials [\[22](#page-12-3)[,23\]](#page-12-2) for water-based lubrication. It is interesting to note that for oil-based lubrication the decrease in the wear rate provided by Ti_3C_2 [\[34,](#page-12-13)[37\]](#page-12-15) is often smaller than that provided by other 2D materials, including the graphene-based materials, h-BN and $MOS₂$ [\[42–](#page-12-19)[45\]](#page-13-2). Nonetheless, the overall results presented in Figure [5,](#page-7-0) along with the data in Figure [3,](#page-4-0) demonstrate the feasibility of $Ti₃C₂$ as a water-based lubrication additive for wear reduction.

To gain a better understanding of the effect of $Ti₃C₂$ on wear, the balls and disks were carefully examined before they were cleaned. As examples, the high-magnification LSCM images obtained after experiments with 0 wt%, 1 wt%, 5 wt%, and 7 wt% $Ti₃C₂$ concentrations are shown in Figure [6a](#page-8-0)–d, respectively. From the images in Figure [6a](#page-8-0), along with those in Figures [4](#page-5-0) and [5,](#page-7-0) signs of abrasive and adhesive wear can be observed on both the ball and disk surfaces after the experiment with 0 wt% Ti₃C₂ concentration. However, the disk wear was more severe than the ball wear, which was due to the fact that the hardness of the disk was significantly smaller than that of the ball. In addition, a slight occurrence of abrasive wear was observed on the ball, which may be associated with the hard wear particles at the contact interface [\[46\]](#page-13-3). Figure [6a](#page-8-0) also shows the wear debris adhered to both the ball and particles at the contact interface [46]. Figure 6a also shows the wear debris adhered to both the ball and disk surfaces. It was likely that the wear debris on the ball and disk surfaces was mainly generated ball and disk surfaces. It was likely that the wear debris on the ball and disk surfaces was mainly from the disk, considering the disk wear volume was one or two orders of magnitude larger than the ball wear volume. larger than the ball wear volume. eres at the contact interface [46]. Figure 6a also shows the wear debris adhered to both the ball

Figure 6. LSCM images of the flattened area of the ball and the wear track of disk after experiments with (a) 0 wt%, (b) 1 wt%, (c) 5 wt%, and (d) 7 wt% Ti_3C_2 concentrations under 3 N normal force. **Figure 6.** LSCM images of the flattened area of the ball and the wear track of disk after experiments

Figure 6 shows that the amount of particles on both the ball and disk surfaces generally increased Figure [6](#page-8-0) shows that the amount of particles on both the ball and disk surfaces generally increased with increasing ${\rm Ti_3C_2}$ concentration. These particles were expected to be mostly ${\rm Ti_3C_2}$ given that the wear volume of the ball and disk was a few orders of magnitude smaller than the volume of Ti $_3\mathrm{C}_2$ in the lubricant. An SEM image and EDS spectrum of these particles on the ball surface are provided in Figu[re 7](#page-9-0). The planar shape of the particles and the relatively strong Ti peak clearly indicate that these particles were mainly Ti_3C_2 , as expected. The planar shape of Ti_3C_2 may be advantageous for it to enter the contact interface. It is likely that the Ti_3C_2 flakes on the contact interface could hinder the direct contact between the ball and disk, leading to suppression of wear progression [\[34\]](#page-12-13). $\frac{1}{2}$ was expected to be more significant at the edges of the edges of the contact area, as discussed to be $\frac{1}{2}$ was discussed to be edges of the contact area, as discussed to $\frac{1}{2}$ was discussed to $\frac{1}{2}$ Particularly, this behavior of Ti_3C_2 was expected to be more significant at the edges of the contact

area, as discussed earlier. It can also be seen that $Ti₃C₂$ was relatively more scattered on both the ball and disk surfaces with 5 wt % concentration (Figure [6c](#page-8-0)) than for 1 wt% concentration (Figure [6b](#page-8-0)). This may be responsible for more significant wear reduction as $Ti₃C₂$ concentration increased up to 5 wt%. However, it can be seen from Figure [6d](#page-8-0) that Ti_3C_2 was more agglomerated after the experiment for 7 wt% Ti_3C_2 concentration. Such agglomeration can be clearly observed from the SEM images of Ti₃C₂ on the ball surface after the experiment with various Ti₃C₂ concentrations, shown in Figure [8.](#page-10-0) It can be seen from the figure that the degree of agglomeration was significantly greater at 7 wt% Ti₃C₂ concentration than at 1 wt%–5 wt% Ti₃C₂ concentrations. It is plausible that more significant agglomeration of Ti_3C_2 at its high concentration may have been responsible for the reduction of the wear suppression effect, as proposed in a previous study [\[42\]](#page-12-19). Figure [9](#page-10-1) shows SEM images of T₁₃C₂ and a tribo-film formed on the ball surface after experiments with 7 wt% $Ti₃C₂$ concentration under a 10 N normal force. The data in Figure [9](#page-10-1) suggest damage of $Ti₃C₂$ and the formation of a tribo-film under relatively high normal force, which may contribute to suppressing the wear progression, as discussed earlier. It should further be noted that Ti_3C_2 can readily be oxidized in water, especially from the edges of the flakes, and TiO₂ can be formed [\[47,](#page-13-4)[48\]](#page-13-5). Hence, as a result of oxidation, the layered structure of $Ti₃C₂$ could be degraded, which may lead to a further decrease in its performance as an additive. However, such a performance decrease could be partly compensated, given that TiO₂ may be used as a lubricant additive [\[17\]](#page-11-13).

Figure 7. Examples of (a) SEM image of Ti₃C₂ and (b) EDS spectrum of Ti₃C₂ on the flattened area of the ball. the ball.

It should be noted that the tribological characteristics may vary significantly, depending on the material pair, and operating and environmental conditions. Hence, studies across a broad spectrum of experimental parameters are needed to elucidate the feasibility of $Ti₃C₂$ as an additive for water-based lubrication. In particular, a database to determine the optimal concentration should be established, given that optimal concentrations of lubricant additives may be dependent on the tribological system. Furthermore, the effects of size, thickness, and functional group on the tribological performance of Ti₃C₂ should be elucidated. In addition, the influence of dispersibility of T_{i3}C₂ on the tribological characteristics should be explored to improve the performance of water-based lubrication. Long-term tests are further needed for the practical applicability of Ti_3C_2 as a lubricant additive. Nevertheless, the results of this work provide useful information for the fundamental tribological properties of Ti_3C_2 as a feasible water-based lubrication additive.

 (a)

1 wt% Ti_3C_3

 (b)

Figure 8. SEM images of the agglomerated Ti_3C_2 on the flattened area of the ball after experiments with (a) 1 wt\% , (b) 2 wt\% , (c) 3 wt\% , (d) 5 wt\% , and (e) 7 wt\% Ti_3C_2 concentrations under 10 N normal force.

Figure 9. SEM images of (a) Ti_3C_2 and (b) tribo-film on the flattened area of the ball surface after experiment with 7 wt% $Ti₃C₂$ concentrations under 10 N normal force.

4. Conclusions

In this work, the friction and wear characteristics of Ti_3C_2 as an additive for water-based lubrication were experimentally assessed using a ball-on-disk tribotester. The experiments were performed using SS balls and disks at varying normal forces and $Ti₃C₂$ concentrations. The overall results show that $Ti₃C₂$ can provide friction and wear reduction for water-based lubrication. The decrease in the direct contact between the ball and the disk offered by Ti_3C_2 flakes was believed to be responsible for this outcome. It was also shown that the hindering of contact was more significant at the edges of the contact interface, which in turn led to suppression of wear progression. Additionally, the degrees of reduction of friction and wear were found to increase from 7% to 20%, and from 20% to 48%, respectively, as Ti_3C_2 concentration increased from 1 wt% to 5 wt%, compared to the friction and wear without Ti₃C₂ additive. However, the reductions in friction and wear were limited to 10% and 23%, respectively, when $Ti₃C₂$ concentration further increased to 7 wt%. The excessive agglomeration of $Ti₃C₂$ flakes may have been responsible for this outcome. The results of this work may provide useful information for a fundamental understanding of the tribological properties of $Ti₃C₂$ as a water-based lubrication additive, and may therefore aid developing environment-friendly lubricants.

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