



Morphological intelligence counters foot slipping in the desert locust and dynamic robots

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During dynamic terrestrial locomotion, animals use complex multifunctional feet to extract friction from the environment. However, whether roboticists assume sufficient surface friction for locomotion or actively compensate for slipping, they use relatively simple point-contact feet. We seek to understand and extract the morphological adaptations of animal feet that contribute to enhancing friction on diverse surfaces, such as the desert locust (*Schistocerca gregaria*) [Bennet-Clark HC (1975) *J Exp Biol* 63:53–83], which has both wet adhesive pads and spines. A buckling region in their knee to accommodate slipping [Bayley TG, Sutton GP, Burrows M (2012) *J Exp Biol* 215:1151–1161], slow nerve conduction velocity (0.5–3 m/s) [Pearson KG, Stein RB, Malhotra SK (1970) *J Exp Biol* 53:299–316], and an ecological pressure to enhance jumping performance for survival [Hawlena D, Kress H, Dufresne ER, Schmitz OJ (2011) *Funct Ecol* 25:279–288] further suggest that the locust operates near the limits of its surface friction, but without sufficient time to actively control its feet. Therefore, all surface adaptation must be through passive mechanics (morphological intelligence), which are unknown. Here, we report the slipping behavior, dynamic attachment, passive mechanics, and interplay between the spines and adhesive pads, studied through both biological and robotic experiments, which contribute to the locust's ability to jump robustly from diverse surfaces. We found slipping to be surface-dependent and common (e.g., wood 1.32 ± 1.19 slips per jump), yet the morphological intelligence of the feet produces a significant chance to reengage the surface (e.g., wood 1.10 ± 1.13 reengagements per jump). Additionally, a discovered noncontact-type jump, further studied robotically, broadens the applicability of the morphological adaptations to both static and dynamic attachment.

locust | robot | slip | friction | jump

Terrestrial locomotion emerges from interaction with one's environment (1), which is further complicated by the diversity of surface materials and dynamic timescales. Locomotion studies have explored dynamics with point-contact feet (2–10), rigid, rough (11–13), mesh (14), and smooth (15, 16) surfaces, deformable (17) surfaces, granular (18–25) surfaces, and obstructed environments (26, 27) elucidating traversing behaviors, optimal strategies, and new environmental material models (28). However, interactions tend to dissipate energy, which is well studied for granular surfaces [environmental dissipation (28)], but not for rigid surfaces (body dissipation) which, through slipping, tend to dissipate energy in the tissues (29). Scansorial robots using either spines (30–32) or gecko-inspired dry adhesives (33–36) have demonstrated the frictional properties of each of these attachment mechanisms on rigid surfaces. However, the diversity of surfaces in the environment sometimes necessitates multiple attachment mechanisms, and, whereas humans have developed shoes for ice, grass, wood, rock, track, slip, and stick, animal feet must be multifunctional.

Desert locusts (1, 37–40) [*Schistocerca gregaria*; female, 2.32 ± 0.31 g; male, 1.67 ± 0.14 g; $t(30) = 10.0$, $P = 0.00$] escape-jumped (41) from an elevated platform in response to a rapid approach from behind by the experimenter (Fig. 1A, Movie S1, and *Animals and Statistics*). The surface material is one of five selected,

including hydrophobic glass (contact angles of 94.7°/84.0°, $n = 119$), hydrophilic glass (contact angles of 32.6°/21.6°, $n = 111$), wood (sawn pine, $n = 113$), sandstone ($n = 112$), and mesh (steel, $n = 95$), where n represents the number of trials (Fig. 2, Table 1, and *Surface Preparation*). These materials were selected to both challenge and isolate the two major attachment mechanisms of the locust: the spines, whose contribution to surface friction is unknown, and wet adhesive pads (Fig. 1B). We proposed that the glass would sufficiently isolate the adhesive pad behavior and the change in wettability would challenge the wet adhesion, whereas the wood, sandstone, and mesh would sufficiently isolate the spine behavior, and the change in surface roughness and cohesion would challenge the spine friction (42). We recorded the foot behavior for each material with a high-speed camera; further details are presented in *Locust Experiments*. Initial observations of the jumping angles (Fig. 1C) and accelerations (Fig. 3A) suggest that the locust's feet exhibit frictional properties beyond that predicted by the traditional Coulomb friction model.

To quantify the performance of the locust's feet on each surface, individual slipping events were counted and categorized by whether the foot reengaged the surface after the slip as (type 1 or 2) slips and the potential energy lost as (planting, early, middle, or late) slips (Fig. 4). The slip-type categories include type-1 slips, where all remaining energy is lost, and type-2 slips, where a slip and reengagement occur, resulting in minimal energy lost (Fig. 3B and C); reengaging the surface occurs when the foot comes to rest after a slip event. The potential energy lost divides

Significance

Animals in natural environments necessarily come into contact with surfaces having differing orientations and materials. However, not much is known about how their feet contribute to friction, particularly in the case of dual-attachment mechanisms and passive mechanics (morphological intelligence). We study the desert locust's (*Schistocerca gregaria*) morphology, spines and adhesive pads, and jumping behavior to extract traits that contribute to enhancing friction. Our results demonstrate the potential contribution of morphological intelligence to solving complex dynamic locomotion problems. We anticipate that this study will inspire further research of the strategies used by animals to interact dynamically with diverse surfaces. Furthermore, the concepts presented can be easily adapted to, for the enhancement of, existing simple miniature and state-of-the-art large-legged terrestrial robots.

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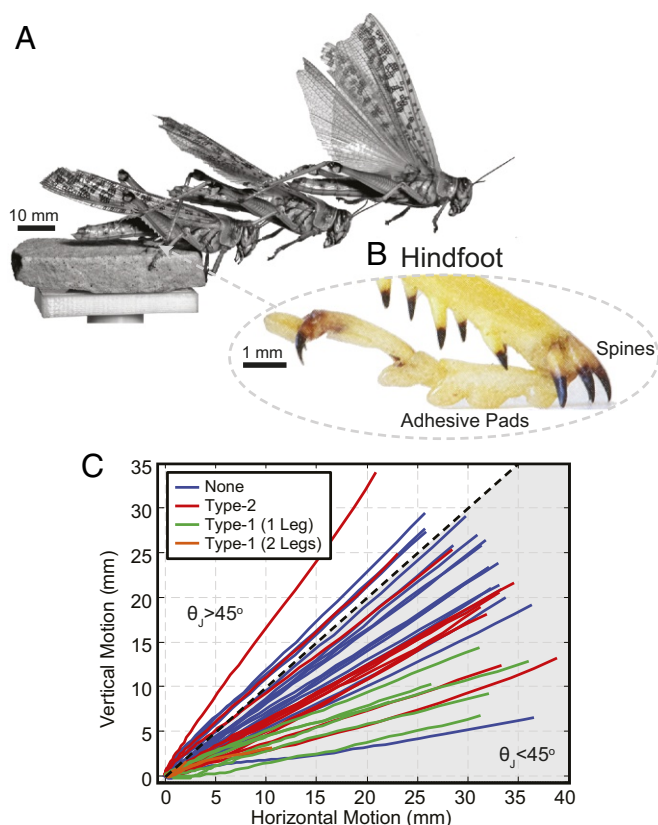


Fig. 1. The desert locust's jumping behavior. (A) Side-view video snapshots of a desert locust (*S. gregaria*) jumping trial. (B) Photo of the locust's hindfoot. (C) Body XY-motion during jumping trials where the locust had either no (none), type-2, type-1 (one leg), and type-1 (two legs) slips (males only; *Trial Details*). Tracking stops when the last foot leaves the surface. Jumping angles, $\theta_j < 45^\circ$ (gray region), require friction coefficients > 1 , which are generally not achievable in the traditional Coulomb friction model.

the slip events into four energy ranges, defined by knee angle, θ_k , ranges at the onset of the slip, which represent the potential energy loss of $\theta_k[0]^\circ \approx 100\%$, $\theta_k[0, 45]^\circ = [100, 38]\%$, $\theta_k[45, 135]^\circ = [38, 0.6]\%$, and $\theta_k[135, 180]^\circ = [0.6, 0]\%$, respectively, where the potential energy fractions come from the kinematics of the robotic platform. Late slips will henceforth be excluded from analyses, as the remaining energy is minimal, and the robotic platform dynamics predict lift-off within this region. The planting slip comes from the discovery that locusts occasionally initiate a jump without their feet in contact (noncontact jump) with the surface, or without sufficient contact, and sometimes a slip occurs before the foot comes to rest, compared with a jump in which the foot begins in contact with the surface (contact jump). To investigate the importance and modification of morphological traits difficult to investigate on the locust, robotic feet were developed for the robotic platform [MultiMo-Bat (43, 44)], which uses a similar catapult-type jump and leg structure to that of the locust (Fig. 6A, *Robotic Foot*, and *Movie S2*).

Results and Discussion

We reasoned that slipping would be common, yet the high probability would suggest very low robustness if not for the discovered possibility to reengage the surface (overall behavior, Fig. 4A; statistical data, Fig. 4B). This discovery, along with that of the noncontact jump behavior (*SI Appendix*, Fig. S2A), suggests that the morphological adaptations of the locust's feet are designed for both static and dynamic surface interactions. The lower

chance of type-1 slips on the rough surfaces (wood, 0.22 ± 0.44 ; sandstone, 0.24 ± 0.47 ; and mesh, 0.0 ± 0.0) compared to the smooth surfaces (hydrophobic glass, 0.83 ± 0.66 ; and hydrophilic glass, 0.88 ± 0.70) further suggest specialization for spine-based friction. However, whereas early and middle type-1 slips showed pervasive significant differences between the rough and smooth surfaces, the planting type-1 slips did not, suggesting that the two friction mechanisms have equal difficulty interacting dynamically with surfaces (Fig. 4B).

Spines. The locust's spines protrude from the distal end of the tibia, defined by two characteristic angles measured from the leg's sagittal and frontal planes (Fig. 5A, *SI Appendix*, Fig. S1A and B, and Table 2), potentially adapted from those seen on the forefeet (*SI Appendix*, section S1) and used for rapid running on challenging surfaces (14). Spines are able to enhance friction by interacting with asperities, which have differing angles than that of the overall surface, effectively increasing the loading angle, θ_{SL}^* , to a more surface-perpendicular orientation. We developed a nondimensional model of the spine tip ($r_s = 23.3 \pm 13.4 \mu\text{m}$) and asperity radii, r_a , which predicted that smaller asperities (mean area roughness: wood, $S_a = 10.6 \mu\text{m}$; sandstone, $S_a = 75.13 \mu\text{m}$) should be less stable (Fig. 5B and *Spine Interaction*), which we reasoned would result in higher slip rates and was confirmed by the locust experiments, in which a significant difference was observed between the mean type-2 slips for wood (1.10 ± 1.13) and sandstone (0.56 ± 0.89), $t(212) = 4.0$, $P = 0.00$ (Fig. 4). However, the similar chance of type-1 slips on wood (0.22 ± 0.44) and sandstone (0.24 ± 0.47), $t(222) = 0.3$, $P = 0.74$, suggested that the locust's spines are well developed to find and hold asperities on a broad range of roughnesses and dynamic conditions, resulting in minimal loss of jumping energy.

The spines may be able to penetrate the soft surface of the wood to enhance or create an asperity; however, the decreasing slip rates on increasingly rough surfaces [wood, 1.17 ± 1.17 , vs. sandstone, 0.59 ± 0.90 , $t(210) = 4.2$, $P = 0.00$; sandstone vs. mesh, 0.02 ± 0.14 , $t(118) = 6.6$, $P = 0.00$] and failure to identify any spine indentations on the wood surface using a 3D surface profilometer suggested that the spines are instead adapted to hold existing asperities. This conclusion was further supported by the discovery of a two-axis passive joint (Fig. 5A), which connects

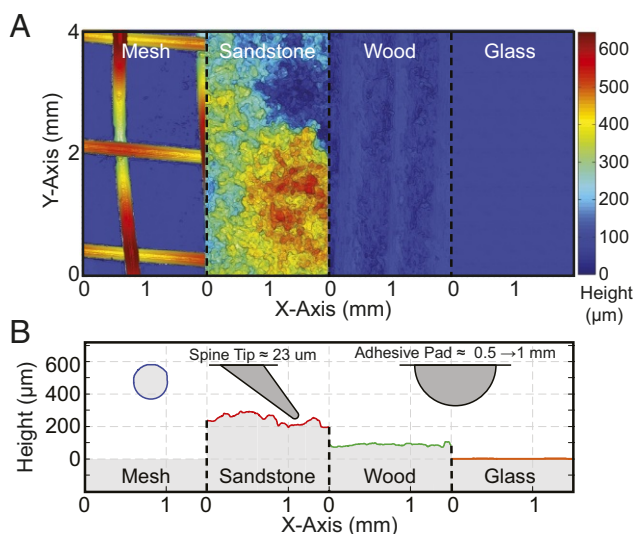


Fig. 2. Experiential surfaces compared with the locust. (A) Surface materials (3D surface profilometer micrographs). (Magnification: $100\times$.) (B) Surface cross-sections compared with the locust's spines and adhesive pads.

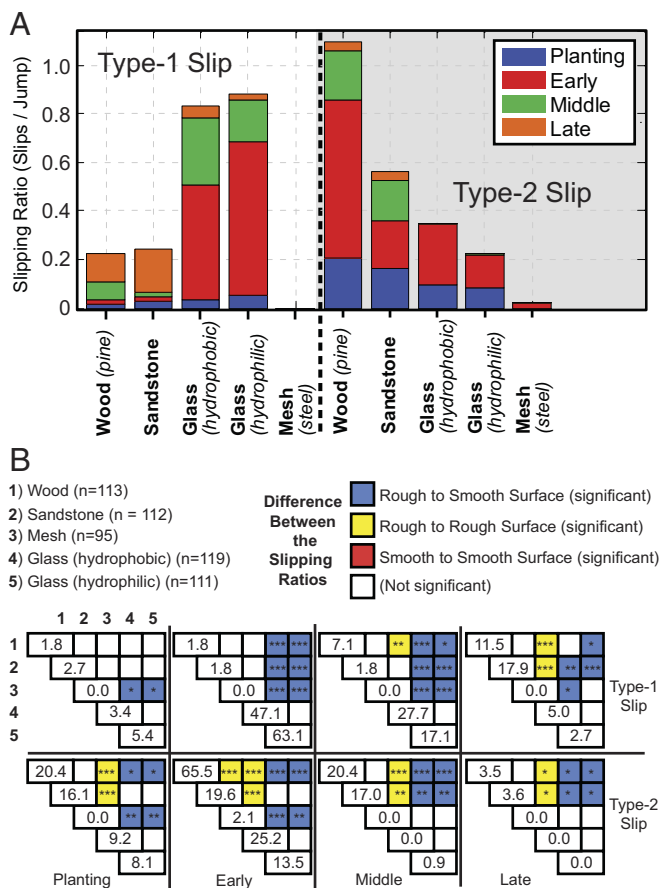


Fig. 4. Locust experimental results per jump (per-leg rates are half), divided into slip type (type 1 or 2) and energy regions (planting, early, middle, or late). (A) Average slipping ratios (slips/jump) for each slip type on each material. (B) Material significance matrix, which compares the slipping ratios of each material within a single slip type and energy region of A, to determine significance of the differences in the means [e.g., type-1 slip and planting (upper left quadrant) presents the 10 unique significance measures between all five materials]. The diagonal values represent the slipping ratios (slips/jump), where zero values indicate that no slipping events were observed. * $P \leq 0.05$; ** $P \leq 0.01$; *** $P \leq 0.001$. n represents the number of trials on each material.

to enhance surface friction on both smooth and rough surfaces. This was confirmed by robotic experiments (Fig. 6 B and C and *Robot Experiments*) which resulted in no type-1 slips (Fig. 6D, SLIP, and *Movie S4*). Foot tracking also related much of the type-2 slips to the zero initial foot loading (preload = 0 N), causing planting type-2 slips during the contact and loading period, and variation in mechanism friction, causing early and middle type-2 slips (e.g., Fig. 6E, symmetric, and *Robot Experiments*).

The non-contact-type jumps, observed in the locust, suggest that the attachment mechanisms are developed for both static and dynamic attachment. This was verified by robotic experiments where the noncontact jumps were accentuated to a misalignment angle of 45° and impact velocities of approximately 3.4 m/s (Fig. 6 C and E, symmetric), resulting in no type-1 slips (Fig. 6D, noncontact); type-2 and -3 (Type-3 Slip) slipping behaviors were slightly increased by the increased dynamics. Furthermore, the constituent parts of the SLIP foot, tested independently (Fig. 6D, SLIP-none, -pad, and -spine), showed the adhesive pad as the primary attachment mechanism on all surfaces and jump conditions, as it was essential in reducing the spine loading, facilitating spine friction on surfaces with low cohesion.

Type-3 Slip. First observed in the locust (*SI Appendix, Fig. S2B* and *Movie S3*), type-3 slips are that of continuous slipping (Fig. 6C), which occurs when the surface-interaction strength is less than the applied load. Only a few occurrences were observed in the locust and only on the glass surfaces, whereas the robot showed a much higher chance of type-3 (contact, 0.60 ± 1.05 ; noncontact, 1.60 ± 0.84) slips, albeit only on the sandstone surface (Fig. 6D). This suggests that while the maximum forces on the locust's spines and adhesive pads were both within the limits of the surface materials' interaction strength, the significant increase in the foot forces of the robot were not. Therefore, requiring the robot to share the load between several spines (30, 45) to reduce the individual interaction forces to that supported by the cohesion of the sandstone grains.

Detachment. Detachment from the surface is accomplished passively for both mechanisms in the locust and robot. Nonembedded spines simply escape the asperities (e.g., sandstone), whereas embedded spines [e.g., wood (robot only)] have been observed to perturb the foot slightly at liftoff, indicating that a small force was required for removal of the spines. The adhesive pads of both the locust and robot, unlike the gecko's highly discretized fibrillar adhesives (46), have relatively large, continuous surfaces, limiting intimate surface contact and the adhesive's ability to resist the propagation of cracks through the interface. The locust was observed to use circumferential peeling of the approximately spherical pads (Fig. 5A), whereas the robot's large rectangular pad (Fig. 6A) was observed to peel from front to back. In both cases, once peeling was initiated, it quickly propagated through the entire surface, passively separating the two, unlike the gecko, which must actively detach its feet (35).

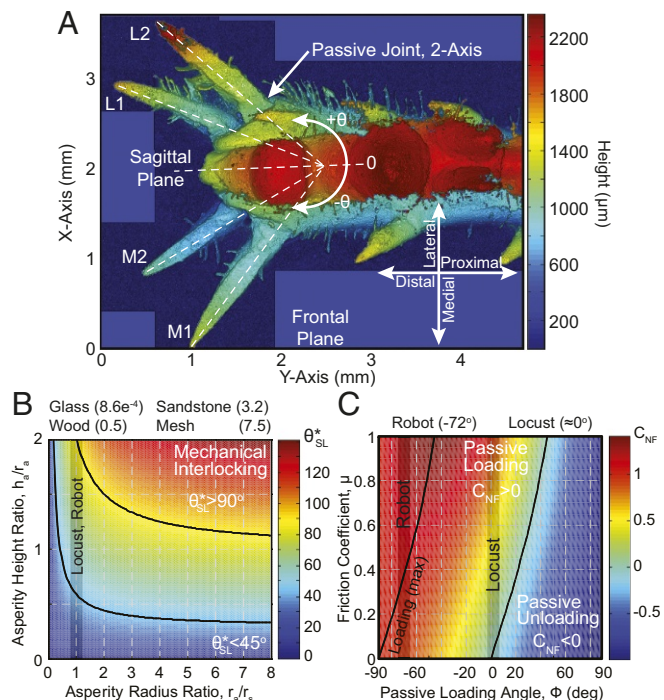


Fig. 5. Locust friction mechanisms. (A) The locust's foot with spines and adhesive pads (3D surface profilometer micrograph). The passive two-axis joint allows for nearly 180° in the sagittal plane and a difference of -10° in the frontal plane. (Magnification: $100\times$.) (B) Spine-asperity interaction plot which determines the potential increases in the effective loading angle θ_{SL}^* as a function of the spine tip radius, r_s , asperity radius, r_a , and asperity height, h_a . The average asperity radius ratios, r_a/r_s , of the tested materials are listed above the plot. (C) Adhesive pad interaction plot, which determines the passive loading coefficient, C_{NF} .

Table 2. Spine characteristics

Spine	Angles to leg planes*	
	Sagittal plane, °	Frontal plane, °
M1	-136.8 ± 9.1	41.0 ± 7.9
M2	-155.3 ± 9.5	42.6 ± 12.6
L2	144.1 ± 8.0	47.7 ± 8.4
L1	126.3 ± 10.1	51.2 ± 8.8

L, lateral; M, medial.

*The angles are measured from the specified planes.

but also bounced off the surface after the initial contact (Fig. 6E, asymmetric). This was due to a shorter lever-arm, which resulted in increased foot rotational rates (symmetric ≈ 160 deg/s, asymmetric ≈ 370 deg/s). This suggests that, although the locust's foot can plant from short distances, the orientation must be similar to that of the surface, whereas the robot's symmetric foot morphology can tolerate much greater misalignment.

Rotational Decoupling. The positioning of the spines and adhesive pads on separate segments, thus decoupling their rotational motions, is integral to the performance of the combined attachment mechanisms. This was confirmed by robotic experiments of several feet where the spines and adhesive pad rotations were coupled and showed significant type-1 slipping (coupled, *SI Appendix*, Fig. S4; decoupled, Fig. 6). This was potentially caused by the passive loading torque of the adhesive pad, which tends to pull the spines away from the surface; in this case, stiffer thermal plastic pads were used to minimize foot rotation during loading. The locust's spine orientations were coupled to the leg angle (*SI Appendix*, Fig. S1B), whereas the robotic SLIP foot had further decoupled the spines, adhesive pad, and leg. This allowed both the spines and adhesive pad to continuously and independently reorient to maintain a desired configuration (Fig. 6F) to passively support dynamic terrestrial locomotion on diverse surfaces.

Jumping Performance. Type-2 slips result in minimal energy loss (Fig. 7A and B, type-2), as evidenced by the similar normalized translational jumping energies (E_N) of the no-slip (none) and type-2 slip cases of the locust, $t(27) = 1.26$, $P = 0.22$, and robot, $t(9) = 0.09$, $P = 0.93$. Type-3 slips showed similar minimal energy loss in the robot, $t(3) = 1.47$, $P = 0.24$; however, these results represented relatively slow foot-sliding velocities, whereas higher velocities may remove significant energy and are left as future work. Unlike the previous, single type-1 slips resulted in significant energy loss [Fig. 7A and B, type-1 (one leg)], as evidenced by the significant difference in the E_N of the no-slip (none) and type-1 (one leg) slip cases of the locust, $t(14) = 7.86$, $P = 0.00$, and robot (calculated, *Slip Energy*).

Type-1 slips may also have an added effect on the jumping performance as they break the sagittal-plane symmetry, common to most animals, introducing a torque on the body (Fig. 7C), as only a single leg is in contact. This was experimentally confirmed by the $>50\%$ reduction in the locust's E_N , observed between the early stage type-1 (one leg) and no-slip (none) cases (Fig. 7A). A simplified model (Fig. 7C and D and *Slip Energy*) captured the performance degradation and demonstrated the significance of the slip's position in the jumping cycle. Earlier slips resulted in not only more energy loss, but more energy converted into rotational, instead of translational, motion, the effect of which was further accentuated by the distance between the leg and center of mass (\approx body width/2).

In total, each of the locust's feet can support an average shear force of $F_{A\parallel} \approx 149$ mN ($F_{A\perp} \approx 131$ mN), whereas the robotic SLIP foot was verified to an average shear force of $F_{A\parallel} \approx 7.7$ N ($F_{A\perp} \approx 7.7$ N) on the rough and smooth surfaces. The

robot experiments verified the attachment mechanisms to a misalignment, or jump, angle of 45° , at which point attachment must rely on more than Coulomb friction. However, with decreasing jump angle ($< 45^\circ$) comes increasing burden on the attachment mechanisms ($F_{A\parallel}(\max) = F_{A(\max)} \cos(\theta_J)$), captured by the adhesive and spine-interaction models, requiring enhanced adhesion and spine interaction to maintain attachment.

Summary

This study highlights slipping not as an anomaly, but as part of the dynamic locomotion of the desert locust, yet, in the absence of control, morphological intelligence encoded into the physical system can reduce the severity of many of the slipping events and preserve robust locomotion on diverse surfaces.

Materials and Methods

Animals. The desert locusts (*S. gregaria*) were obtained from a local pet store and housed in a terrarium with heat lamp and plentiful food supply. Any stress or nutritional deficiency was abated by an acclimation period of no less than 5 d, where no experimental trials were conducted. Grass was the main source of food provided to the locusts.

Locust Experiments. The desert locusts were transferred from their main terrarium to an empty acrylic terrarium with a heat lamp and thermometer. The locusts, being cold-blooded, were warmed up to an average temperature of 30.5°C before being transferred to the testing terrarium. This process also allowed them to clean their adhesive pads, especially important before smooth-surface trials.

An acrylic box was constructed to contain the locusts after each jump, while allowing unrestricted viewing of the event. In the center, a small platform was placed, on which the surface could be modified for testing various parameters of the locust's jump. The platform dimensions were specifically selected to encourage the locust to align itself perpendicular to the viewing direction. The high-speed digital camera (VisionResearch, Phantom v641) recorded the jumping trials at 1,400 frames per second.

Surface Preparation. The hydrophilic glass was prepared by cleaning a glass slide with varying polarity solvents; the sequence was as follows: acetone, isopropyl alcohol, deionized water, and argon gas for drying. To fabricate the hydrophobic glass, the surface chemistry of some of the cleaned glass slides was altered through a silanization process as follows: air plasma, immersion in butylamine and methoxy(dimethyl) octadecylsilane, and condensation at 80°C . The advancing and receding contact angles were measured by using a Krüss-Drop Shape Analyzer. The wood, sandstone, and steel mesh were purchased from a local hardware store and cut to proper size. The wood was smooth-cut pine, chosen for its relative softness.

Imaging. The surface and locust foot morphology plots were generated with a high-resolution 3D surface profilometer (Keyence VK-X200) with x -axis and y -axis resolutions of 1,390 nm and z -axis resolutions of 0.1 nm.

Trial Details. The selected jump trials each had only a single slip event occurring in the planting or early knee angle ranges, as these would produce the most observable and similar effects in the jumping energy for a given singular slip event. These approximated the worst case for each slip type, where n represents the number of trials.

Figs. 1C, 3A, and 7A had the following surface breakdown per slip type: none: hydrophilic glass ($n = 5$), sandstone ($n = 6$), and mesh ($n = 6$); type 2: hydrophobic glass ($n = 1$), hydrophilic glass ($n = 2$), wood ($n = 10$), and sandstone ($n = 3$); and type 1 (one leg): hydrophilic glass ($n = 4$) and sandstone ($n = 2$), including planting ($n = 1$), early ($n = 3$), and middle ($n = 2$). The middle trials were just after the transition from the early region: type-1 (two leg): hydrophilic glass ($n = 6$).

Fig. 7B had the following surface breakdown per slip type: none (C): hydrophilic glass ($n = 6$), wood ($n = 9$), and sandstone ($n = 4$); none (N): hydrophilic glass ($n = 4$), wood ($n = 4$), and sandstone ($n = 1$); type 2: hydrophilic glass ($n = 5$), wood ($n = 1$), and sandstone ($n = 1$); type 3: sandstone ($n = 3$); and type-1 (one leg): hydrophilic glass ($n = 1$), wood ($n = 1$), and sandstone ($n = 1$).

Statistics. The reported statistical values were obtained from two-sample, two-tailed t tests, where significance, p , was defined as $P \leq 0.05$ with a null

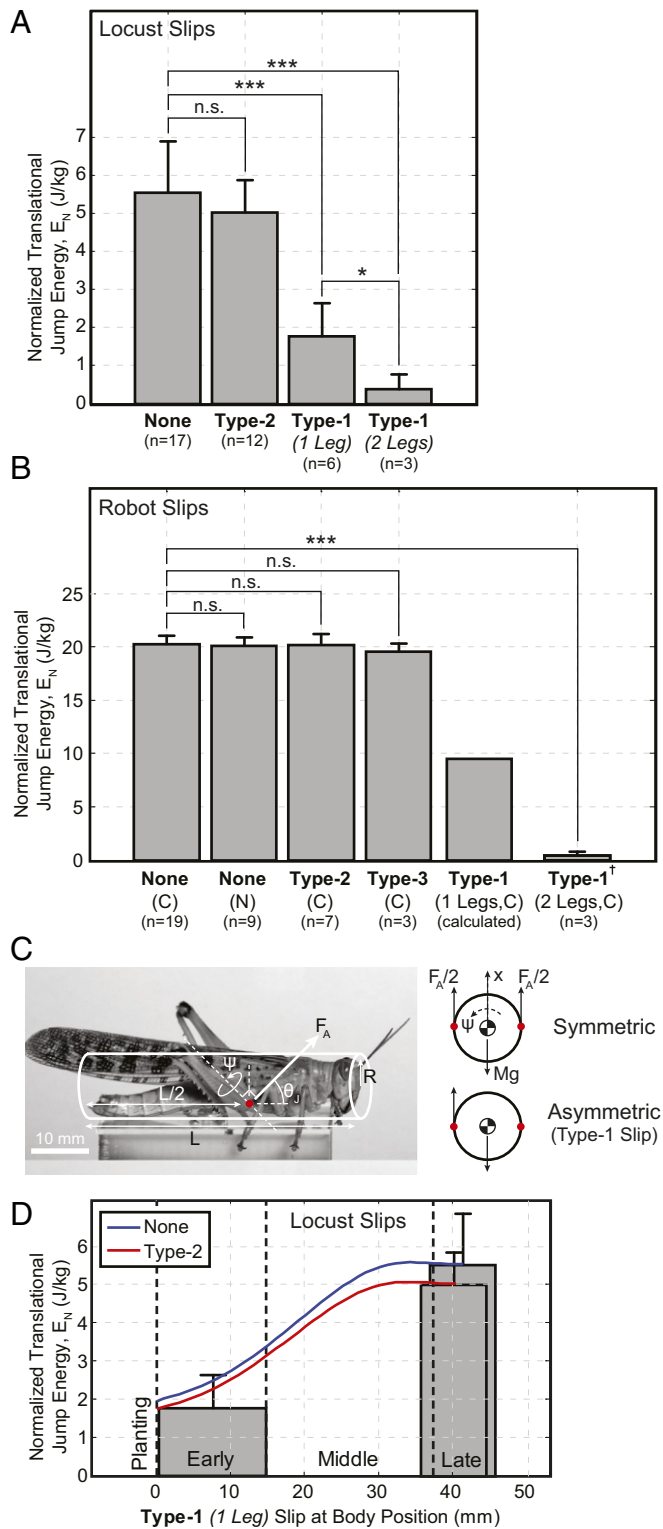


Fig. 7. Mass normalized translational jumping energy (E_N) for different types of slips (*Trial Details and Slip Energy*). (A) Locust: males only. (B) Robot: SLIP foot at a 45° surface angle. † Data are from the SLIP-none foot. (C) Simplified model (*Slip Energy*) for determining the ratio of translational and rotational energies in the event of a type-1 slip. (D) Locust: estimated E_N as a function of the position, within the jumping cycle, in which a type-1 slip occurs. Assumed jumping angle is 45° . Energy regions (planting, early, middle, or late) are shown for an approximate leg length of 20 mm. Approximate position for the included type-1 (one leg) slip data from A is shown. Notes: Work is measured from jump initiation to foot separation, where each slip type is composed of slips from approximately the early knee

hypothesis of equal mean values. The effective degrees of freedom were approximated with the Welch–Satterthwaite equation, and all reported \pm values represent a SD. All reported statistical data follow the following template: $t(n1) = n2, P = n3$, where $n1$ is the effective degrees of freedom, $n2$ is the t -critical value, and $n3$ is the significance or P value.

Spine Orientation. Comparing the sagittal-plane angles of the corresponding medial (M) and lateral (L) spines, M1, L1, $t(18) = 2.44, P = 0.03$, and M2, L2, $P = 0.03$ and $t(17) = 2.86, P = 0.01$, showed low significant differences in the angles, whereas the frontal-plane angles only showed a significant difference, albeit low, between M1 and L1, $t(18) = 2.85, P = 0.01$, indicating only a minor probability of spine asymmetry. However, the angles did indicate that all of the spine would be oriented to interact with the surface during jumping (Table 2).

SI Appendix, Fig. S1B shows the relationship between leg, θ_L , spine, θ_S , jumping, θ_J , and spine loading, $\theta_{SL} = \theta_J$, angles, where $\theta_S \leq 0$ indicates that the spine tip is no longer able to interact with the surface. The passive joints which connect the spines to the tibia give them the freedom to change their sagittal-plane angle from the initial positions (Table 2) to nearly 180° (SI Appendix, Fig. S1B, SP = 180°). This suggests that the spines can passively locate stable positions on the surface as they are loaded. These joints also allow for approximately -10° of rotation in the frontal plane (SI Appendix, Fig. S1B, FP- 10°). Therefore, as the spines are loaded, their effective spine angle is reduced, broadening the leg angle range over which they can interact with the surface. This joint will also include some damping, which can be used to reduce the impulse caused by the spine reengaging after a slip, increasing the chance of a successful reengagement. In all, locusts have developed spines that can interact with the surface even in extreme cases; for example, a horizontal jump ($\theta_J = 0^\circ$ and initial $\theta_L = 90^\circ$) resulted in spine angles of $\theta_S \in [38.8^\circ, 59^\circ]$, permitting strong surface interaction. Spine placement, however, typically only allows the more distal medial and lateral spines to interact with the surface.

Spine Interaction. The performance of spines is strongly related to the roughness characteristics of the surface, spine angle θ_S , and spine loading angle θ_{SL} . We have included surfaces with overhanging asperities and, for our purposes, will calculate an effective increase to the spine loading angle for round asperities as:

$$\theta_{SL}^* = \cos^{-1}(1 - (P_p P_s)/(1 + P_s)), \quad [1]$$

where $P_s \in [0, \infty]$ is the asperity radius ratio, r_a/r_s , and $P_p \in [0, 2]$ is the asperity height ratio, h_a/r_a , with asperity radius, r_a , asperity height, h_a , and spine radius, r_s (Fig. 5B). Mechanical interlocking occurs at $\theta_{SL} + \theta_{SL}^* \geq 90^\circ$, whereas $\theta_{SL} + \theta_{SL}^* < 90^\circ$ reduces the required friction coefficient; for example, $S_a(\text{wood}) = 10.6 \mu\text{m} \Rightarrow P_s = 10.6/23.3$ results in effective increase in the spine loading angles of $\theta_{SL}^*(P_p = 2) = 68^\circ$ and $\theta_{SL}^*(P_p = 1) = 46.6^\circ$ requiring a friction coefficient $\mu = 0.40$ and $\mu = 0.95$, respectively, for a horizontal jump ($\theta_J = 0^\circ$). A friction coefficient of $\mu = 0.40$ is within generally achievable limits, demonstrating that even at extreme jumping angles, the locust's spines greatly enhance the surface friction. We expect the spines to find asperities with increasing likelihood on the wood, sandstone, and mesh. The glass, however, does not produce any usable asperities, and therefore the adhesive pad performance can be isolated.

Adhesive Pad Interaction. Using the morphology (SI Appendix, Fig. S1A) and summing the torques about pad P1 yields an expression for the torque on segment S1 as follows,

$$\begin{aligned} M_A + F_A \sin \theta_J r_A (\mu \cos \phi - \sin \phi) + \mu F_{adh} r_A \cos \phi \\ = R_p r_{2x} + R_p (r_{2x}/r_{3x}) r_{3x} \end{aligned} \quad [2]$$

where F_A is the applied ankle force, M_A is the applied ankle torque, and r_A is the distance between P1 and the ankle joint J_A . Then, R_p is the reaction force at pad P2, $R_p(r_{2x}/r_{3x})$ is the reaction force at pad P3, and r_{2x} and r_{3x} are the x -distances between pads P1-P2 and P1-P3, respectively. The frictional, $\mu F_A \sin \theta_J$, and adhesive, μF_{adh} , forces have been substituted in for the applied horizontal force to limit it to that supported by the surface friction. The left side is then related to the applied loads, and the

angle region, to accentuate the energy losses, and covers all materials in which the behavior was observed. * $P \leq 0.05$; ** $P \leq 0.01$; *** $P \leq 0.001$; n.s., not significant. Variables: number of trials, n ; contact-type jump, C; and non-contact-type jump, N.

right side, the passive loading of the adhesive pads, where the jumping angle range $\theta_j \in [0, 90^\circ]$, $P1\text{-}J_A$ angle range $\phi \in [-90^\circ, 90^\circ]$ ($\phi > 0$ is in the $+x$ -direction), and the friction coefficient μ determine the loading behavior.

The adhesive force, F_{adh} , can be approximated by the pull-off adhesive force of a flat punch from a flat surface. The maximum force is given by (59),

$$F_{adh} = \sqrt{6\pi r_p^3 K W_{adh}}, \quad K = \frac{4}{3} \left[\frac{1 - \nu_p^2}{E_p} + \frac{1 - \nu_g^2}{E_g} \right]^{-1}, \quad [3]$$

where the effective pad radius, r_p , effective work of adhesion, W_{adh} , and the elastic moduli and Poisson's ratios of glass, E_g , ν_g , and the adhesive pad, E_p , ν_p , respectively, determine the behavior. The effective work of adhesion of the locust's pads (6 J/m²) (47) is greater than five times that of the SLIP foot's adhesive pad material (Ecoflex 00-20, 1.17 ± 0.13 J/m², *Work of Adhesion*).

For Eq. 2 to produce a negative torque, unloading the adhesive pads, the normal vs. frictional load coefficient, $C_{NF} = (\mu \cos \phi - \sin \phi)$, must be less than zero, $C_{NF} < 0$, assuming the adhesion is always attractive. Fig. 5C presents the relationship between the friction coefficient, μ , and the $P1\text{-}J_A$ angle, ϕ , where the optimal $P1\text{-}J_A$ angle is $\phi = \tan^{-1}(-1/\mu)$ and the transition between loading and unloading is governed by $\phi = \tan^{-1} \mu$. The highly deformable adhesive pads makes exact measurement of the angle difficult; however, under load, ϕ approaches zero. Applied ankle torque, M_A , can be used to increase the pad loading for additional adhesion, whereas pads $P4$ and $P5$ will generally not contribute.

Work of Adhesion. The work of adhesion was calculated directly from measured pull-off force-displacement curves of the soft elastomer foot pads (Smooth-On, Ecoflex 00-20) by using a 4-mm hemispherical indenter. Three separate sites on the material were tested, where an initial set of trials were conducted to allow the adhesion to stabilize before the measurements were taken. Finally, the work of adhesion was calculated by using the measured adhesive force-displacement curve and contact area. The resulting work of adhesion values are as follows: Ecoflex 00-10 = 1.207 ± 0.036 J/m², Ecoflex 00-20 = 1.168 ± 0.128 J/m², and thermal plastic = 0.142 ± 0.015 J/m², whereas from literature, the locust's work of adhesion was measured as ~ 6 J/m²(47). The locust, however, was measured in a much more idealized configuration. Therefore, the actual value achieved during locomotion may have been much lower, as evidenced by the locust's poor permanence on smooth surfaces.

Robotic Foot. The SLIP foot, Fig. 6A, incorporates all of the features extracted from the locust's foot. The spines are steel pins mounted at a 45° angle and attached to flexural beams [cross-section 1×3 mm ($w \times h$)—material modulus experimentally determined in *SI Appendix, section S3*—which allow sufficient freedom to move both horizontally $\theta_{SH} = \pm 4.8^\circ$ (± 1 mm) and vertically, θ_{SV} , to find asperities and strong enough to penetrate soft surfaces to create its own asperities. The elastomeric adhesive pads (Smooth-On, Ecoflex 00-20) are dry, using instead van der Waals forces to eliminate the difficulty of maintaining a fluid on the feet. The pad is symmetric about the ankle joint, with $\phi \approx -72^\circ$, thus ensuring passive pad loading. The rotational dynamics of the adhesive pad, θ_{PV} , and spines, θ_{SV} , are decoupled from each other as well as the leg, θ_L , allowing each to passively orient to the surface throughout the jump, including during slipping. The loading of the spines and adhesive pad is similar to that of the locust, where the pad must be deformed for the spines to engage the surface. However, as the adhesive pad has a thickness of 1 mm and 100% tensile modulus of 55.2 kPa, initial deformation of the distal region opposite the spines requires little force, allowing the spines to interact immediately. Finally, inspiration is taken from planthopper (*Issus*) nymphs, which use biologically created gears (60), to synchronize the two sides of the robot's legs, ensuring the force produced is along the center-of-mass axis (*SI Appendix, Fig. S5 B and C*).

Foot Forces. The robot's spines and adhesive pad forces are determined from the kinematics,

$$\begin{aligned} F_A &= K_{s1} x_f + K_{p1}(x_f - x_{p0})\delta_p + K_{pb}(x_f - x_{pb0})\delta_{pb} \\ \Rightarrow x_f &= \frac{F_A + K_{p1}x_{p0}\delta_p + K_{pb}x_{pb0}\delta_{pb}}{K_{s1} + K_{p1}\delta_p + K_{pb}\delta_{pb}}, \\ \delta_p &= \begin{cases} 1 & x_f - x_{p0} > 0 \\ 0 & \text{otherwise} \end{cases}, \quad \delta_{pb} = \begin{cases} 1 & x_f - x_{pb0} > 0 \\ 0 & \text{otherwise} \end{cases}, \end{aligned} \quad [4]$$

where each is modeled as simple spring and distance before full pad contact, x_{p0} , distance before backing material engagement, x_{pb0} , and applied load, F_A , dictate the total foot displacement, x_f . The spring constants are,

$$K_{s1} = \frac{6E_{s1} I_{s1}}{2L_{s1}^3}, \quad K_{p1} = \frac{E_{p1} A_{p1}}{L_{p1}}, \quad [5]$$

where the lever arm parameters, including elastic modulus, $E_{s1} = 173.25$ MPa, moment of area, $I_{s1} = 2.76 \times 10^{-12}$ m⁴, and length, $L_{s1} = 0.0182$ m, and the adhesive pad parameters including, elastic modulus, $E_{p1} = 55.16$ kPa, area, $A_{p1} = 2.39 \times 10^{-4}$ m², and thickness, $L_{p1} = 0.001$ m, define the relationships. Finally, the robot's spine, F_{s1} , and adhesive pad, F_{p1} , forces are defined as,

$$\begin{aligned} F_{s1} &= K_{s1} x_f, \\ F_{p1} &= K_{p1}(x_f - x_{p0}) + K_{pb}(x_f - x_{pb0}). \end{aligned} \quad [6]$$

The locust's adhesive pad force is determined by the hertz contact model of two elastic spheres,

$$\begin{aligned} x_p^3 &= \frac{9\pi^2 F_p^2 (R_{p2} + R_{surf})(K_{p2} + K_{surf})^2}{16R_{p2}R_{surf}} \\ \Rightarrow R_{p2} &<< R_{surf} \\ K_{p2} &>> K_{surf} \\ \Rightarrow F_{p2} &= \left(\frac{16R_{p2}x_p^3}{9\pi^2 K_{p2}^2} \right)^{0.5}, \\ K_{p2} &= \frac{1 - \nu_{p2}^2}{\pi E_{p2}}, \quad K_{surf} = \frac{1 - \nu_{surf}^2}{\pi E_{surf}}, \end{aligned} \quad [7]$$

where the radii, $R_{p2} = 290$ μ m and $R_{surf} \gg R_{p2}$, Poisson's ratios, $\nu_{p2} = 0.5$ and ν_{surf} , and tensile moduli, $E_{p2} = 300$ kPa and E_{surf} , for the adhesive pad and surface, respectively, and the assumed full hemisphere deformation, $x_p = R_{p2}$, determine the pad loading. Then, subtraction from the total applied force, F_A , determines the locust's spine loading.

The applied force, F_A , during jumping is calculated through feature tracking (custom Matlab software) for the locust and robot.

Robot Experiments. To experimentally characterize the performance of the SLIP foot, a specific series of trials were conducted at a jumping angle of $\theta_j = 45^\circ$, at which point the necessary friction coefficient is difficult to achieve using the traditional Coulomb friction model. A dynamically similar jumping rig, *SI Appendix, Fig. S5A*, was used to facilitate observation (VisionResearch, Phantom v641) of the foot and uniform testing (*Dynamics*). The testing sequence was as follows: glass, sandstone, wood, and glass, where 10 trials were conducted for, first, the noncontact and, then, contact-type jumps on each surface in the sequence, where the approximate foot impact velocity was 3.4 and 0 m/s, respectively. The order was designed to challenge the SLIP foot, as no modifications or cleaning were conducted between trials; the glass and sandstone could dull or damage the spines, whereas the sandstone and wood could contaminate and damage the adhesive. The order ensured that each material was tested after the SLIP foot had undergone jumps which could reduce the performance of the primary attachment mechanism.

The robot showed much more deterministic jumping behavior than that of the locust (SLIP foot, Fig. 6; coupled foot, *SI Appendix, Fig. S4*). As the preloading of the locust's foot is unknown, the contact jump experiments were conducted at zero preload to simulate the most difficult situation. However, as both attachment mechanisms required some applied load to create friction, an initial slip (planting type 2) was regularly observed during the period before sufficient friction had been created to resist slipping. Type-3 slips were only observed in the sandstone, with higher occurrence in the more dynamic noncontact jumps. These produced a slip of approximately 1.5–2.5 cm, observed as approximately linear scratches in the sandstone, resulting in only a minor loss of energy.

The high occurrence of early type-2 slips for noncontact jumps (Fig. 6) is caused by the energy storage mechanism and not the foot morphology. Fig. 6E demonstrates the typical sudden drop-off in foot velocity which precedes the slip event; this is observed as the foot pulling away from the surface. It is caused by nonuniform friction in the cable mechanism which is accentuated by the sudden velocity changes of the non-contact-type jump, as evidenced by the higher early type-2 slip rates of noncontact jumps. This further challenges the SLIP feet to interact with the surface, and we would

expect improved results if the legs were locked and the cable unwound before the jump.

Finally, the SLIP feet were tested on the MultiMo-Bat (Fig. 6F) to validate the system integration and performance. The MultiMo-Bat was tested first on a flat Acrylonitrile butadiene styrene (ABS) surface and then on angled ($\theta_j = 45^\circ$) glass, sandstone, and wood surfaces, where the behavior matched that of the constrained jumping rig trials (Fig. 6F). Body posture on the angled surface, as with the locust, was maintained through a combination of foot and body contacts. The MultiMo-Bat produced a maximum jumping height of 2.75 m, with similar jumping heights for all cases, where the jumping energy density (J/kg) is approximately equal to the jumping rig. Increasing the jumping energy density by 80% (Fig. 6F) propelled the MultiMo-Bat up to the maximum height of 4.52 m. Continued slipping was observed on the angled wood surface due to the significant increase in tip loading; however, the passive confirmation of the SLIP foot allowed the MultiMo-Bat to extract sufficient energy to propel itself up to heights >4 m.

Dynamics. To analyze the robotic foot's interaction with the surface, a jumping rig was developed which simulates the dynamics of the robot, but within a more observable configuration. The configuration, *SI Appendix, Fig. S5 A and B*, was designed to be dynamically similar to that of the robot, *SI Appendix, Fig. S5C*, which is governed by

$$m_B \ddot{r}_1 = 2k_1 \left(L^2 - \left(\frac{r_1 - r_2}{2} \right)^2 \right)^{0.5} \quad [8]$$

$$m_F \ddot{r}_2 = -2k_1 \left(L^2 - \left(\frac{r_1 - r_2}{2} \right)^2 \right)^{0.5} - k_2 r_2 - c_2 \dot{r}_2,$$

whereas the jumping rig is governed by

$$m_B \ddot{r}_1 = -k_2(r_1 - r_2) - c_2(\dot{r}_1 - \dot{r}_2) \quad [9]$$

$$m_F \ddot{r}_2 = 2k_1 \left(L^2 - \left(\frac{r_2}{2} \right)^2 \right)^{0.5} + k_2(r_1 - r_2) + c_2(\dot{r}_1 - \dot{r}_2),$$

where r_1 and r_2 represent to position of the two masses, main body, $m_B = 55.4$ g, and foot, $m_F = 4.8$ g. The robot's $k_1 = 291.2$ N/m and $L = 135.8$ mm are the stiffness of the main power spring and the leg length, respectively. The ground interaction is governed by $k_2 = 10^9$ N/m and $c_2 = 88.4$ Ns/m, which are the stiffness and damping coefficient, respectively. Note that all values represent half of the robot, as only one leg is being tested. The ground stiffness was chosen to be arbitrarily large compared with k_1 , whereas the damping coefficient c_2 was calculated from the experimentally determined coefficient of restitution, $COR = 0.8929$, of the 3D-printed ABS material (Stratasys uPrintSE Plus, ABSplus), discussed in *SI Appendix, section S3*. The gravitational potential contributed little to the jumping dynamics, as the accelerations were an order of magnitude greater over short periods ($\approx [25, 50]$ ms) (Fig. 3A). Therefore, the orientation of the trials played an insignificant role in the dynamics during the foot contact.

A comparison of the dynamics, *SI Appendix, Fig. S5D*, showed good agreement between the magnitude and damping responses of the testing

rig and the robot, with only a small variation in the bouncing frequency of the foot (*SI Appendix, section S2*).

Slip Energy. The mass normalized translational jump energies, presented in Fig. 7 A and B, were calculated by numerical integration (trapezoidal rule) from the acceleration data. To ensure uniformity, only male trials were included (*Trial Details*).

Assuming the locust as a uniform cylinder, the rotational inertia (I) as a function of jumping angle (θ_j) is calculated as follows,

$$I = \int_{-L/2}^{L/2} \int_{-\pi}^{\pi} \int_0^R \rho r d^2 dr d\phi dl$$

$$\text{where, } d = \ell \cos(\theta_j) - r \sin(\phi) \sin(\theta_j),$$

$$M = \pi L \rho R^2;$$

$$I = \frac{1}{12} M \left(L^2 \cos^2(\theta_j) + 3R^2 \sin^2(\theta_j) \right) \quad [10]$$

where the length of the body (L), effective radius (R), jumping angle (θ_j), effective density (ρ), and mass (M) define the relationship. The variation in the rotational inertia comes from the assumption that a type-1 slip would hinder the locust's ability to properly align its body. The ratio of the translational to total energy, $P_T = E_T / (E_T + E_R)$, caused by a type-1 slip, for the locust is then,

$$P_{TL} = \left(\frac{12 F_A^2 R^2}{(F_A - Mg)^2 (L^2 \cos^2(\theta_j) + 3R^2 \sin^2(\theta_j))} + 1 \right)^{-1} \quad [11]$$

and the robot, having constant rotational inertia (I), is then,

$$P_{TR} = \left(\frac{F_A^2 M R^2}{I(F_A - Mg)^2} + 1 \right)^{-1}, \quad [12]$$

where the translational energy (E_T), rotational energy (E_R), applied force (F_A), and the gravitational constant ($g = 9.81$ m/s²) define the ratios. These allow for estimation of the effects of a type-1 slip (locust, Fig. 7B) at different points of the jumping cycle, from the no-slip (none) cases presented in Fig. 7 A and C as,

$$E_N(a) = \int_0^a \ddot{x} dx + \int_a^b \ddot{x} P_{T,R} dx \quad [13]$$

where the body position at the onset of the type-1 slip (a), body position at liftoff (b), and body position along motion axis (x) define the behavior. Comparing the normalized translational jumping energy of the predicted and actual type-1 (one leg) slips of the locust shows good agreement (Fig. 7D), which allows for approximation of the effect to the robotic platform [Fig. 7B, type-1 (one leg)].

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